

AMENDMENTS TO THE CLAIMS:

This listing of claims will replace all prior versions, and listings, of claims in the application:

Listing of Claims:

Claims 1-4. (Canceled)

5. (Currently amended) Crank shear, especially for cutting rolled strip (22), which comprises two pairs of blades (3, 4) that can be mounted on blade holders (1, 2), wherein the blade holders (1, 2) are supported opposite each other in a vertical plane (x-x) in a pair of eccentric drive shafts (5, 6) and are pivoted on torque supporting levers (7, 8) in double-joint mechanisms (9, 10) in interaction with two hydraulic control units (11, 12) that act on these double-joint mechanisms (9, 10), wherein the blade holders (1, 2) form axially parallel pairs of bearing surfaces (16-19) for the pairs of blades (3, 4) on approximately radial projections (13-15), with the upper pair of blades (3) arranged on inner, facing bearing surfaces (16, 17) of a curved recess (20) of the upper blade holder (1), and with the lower pair of blades (4) arranged on outer, oppositely directed

bearing surfaces (19, 18) of a projection (15) oriented towards the recess (20), the projection being relatively smaller and formed as a single piece with the lower blade holder, each blade of the lower pair of blades having a completely exposed surface facing away from the other blade of the lower pair of blades.

6. (Previously presented) Crank shear in accordance with Claim 5, wherein in a spread position of the torque supporting levers (7, 8) of approximately 90° and at the shortest separation (D) of the eccentric shafts (5, 6) and a running direction (21) of the rolled strip (22) towards the supporting levers (7, 8), a position of the pair of blades (3) for the cropping cut at the leading end (23) of the strip is reached, in which the hydraulic control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully extended, and the control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully retracted.

7. (Previously presented) Crank shear in accordance with Claim 5, wherein, in a spread position of the torque supporting levers (7, 8) of approximately 90° and at the greatest separation (d) of the eccentric shafts (5, 6) and a running direction (21) of the rolled strip (22) towards the supporting levers (7, 8), a

passage position of the crank shear is reached, in which the control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully retracted, and the control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully extended.

8. (Previously presented) Crank shear in accordance with Claim 5, wherein in an approximately horizontal parallel position of the torque supporting levers (7, 8) opposite the running direction (21) of the rolled strip (22) and at the shortest separation (D) of the eccentric shafts (5, 6), and with the upper double-joint mechanism (9) and lower double-joint mechanism (10) extended approximately linearly, a position of the rear pair of blades for cutting the tail end (24) of the strip is reached, in which the hydraulic control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully retracted, and the control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully extended.

9. (Previously presented) Crank shear in accordance with Claim 5, wherein, in a position of the upper supporting lever (7) that is downwardly inclined towards the rolled strip (22) with the upper hydraulic control unit (11) of the double-joint

mechanism (9) retracted, and in a position of the lower supporting lever (8) that is upwardly inclined towards the rolled strip (22) with the lower control unit (12) of the double-joint mechanism (10) fully extended, and with the greatest separation (d) of the eccentric shafts (5, 6), the passage position through the shear is reached.